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1-19. (CANCELED)

20. (CURRENTLY AMENDED) A planetary gear train, comprising:

a ring gear (14),

a sun gear (4), and

~~having~~ a plurality of planetary gears (8) rotatably mounted on a planetary gear carrier (18) by corresponding planetary gear axles (16)[[and]],

wherein each planetary gear (8) is in tooth contact with [[a]] the ring gear (14) and [[a]] the sun gear (4),

the planetary gear axles (16) ~~being retained~~ are mounted at an inclined angle (α) ~~in the planetary gear carrier (18)~~ with respect to an axle (2) of the sun gear (4),

each planetary gear (16) is axially displaceable upon the corresponding planetary gear axle (16), and

the ring gear (14) and the planetary gears (8) are of conical design, and

~~the planetary gears (8) being disposed axially displaceable upon the planetary gear axles (16) coordinated therewith, and~~

an adjusting mechanism is operatively situated between the planetary gear carrier (18) and the planetary gears (8) for adjusting a position of the planetary gears (8) ~~in the planetary gear train~~ on the planetary gear axles (16) for a backlash, ~~the adjusting mechanism being operatively situated between the planetary gear carrier (18) and the planetary gears (8)~~.

21. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the sun gear (4) has an approximately cylindrical external toothing (6).

22. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein a cone angle (β) of tooth flanks of the planetary gears (8) is adapted to an inclined angle (α) of the planetary gear axles (16) so that tooth flanks of the sun gear

PAGE 3/34 * RCVD AT 4/24/2009 4:36:45 PM [Eastern Daylight Time] * SVR:USPTO-EFXRF-5/33 * DNIS:2738300 * CSID:603 226 7499 * DURATION (mm-ss):07-14

(4) and tooth flanks of the planetary gears (8) mesh with one another over a whole tooth width.

23. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 22, wherein the inclined angle (α) of the planetary gear axles (16) corresponds at least approximately to the cone angle (β) of the tooth flanks of the planetary gears (8).

24. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 22, wherein the inclined angle (α) of the planetary gear axles (16) corresponds at least approximately to half the cone angle of the tooth flanks of the ring gear (14).

25. (PREVIOUSLY PRESENTED) The planetary gear train according claim 20, wherein the planet carrier (18) is rotatably mounted in the ring gear (14).

26. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 25, wherein two bearings (24) are provided on both sides of a toothing plane of the planetary gears (8).

27. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 25, wherein bearings (24) between the ring gear (14) and the planet carrier (18) are slanted bearings (24) in an O-arrangement.

28. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the planet carrier (18) is connected with an output shaft of the gear train.

29. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the sun gear (4) is connected with to an input shaft (2) of a prime mover.

30. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the adjusting mechanism, between planetary gear carrier (18) and ring gear (14), comprises at least one of operative fitting discs (32) and spacer discs (34) which determine an axial position relative to each other.

31. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the adjusting mechanism operatively situated between planetary gear

PAGE 4/34 * RCVD AT 4/24/2009 4:36:45 PM [Eastern Daylight Time] * SVR:USPTO-EFXRF-5/33 * DNIS:2738300 * CSID:603 226 7499 * DURATION (mm-ss):07-14

carrier (18) and planetary gears (8) are spacer pieces (58) located coaxially relative to the planetary gear axles (16).

32. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the adjusting mechanism operatively situated between planetary gear carrier (18) and planetary gears (8) are adjusting springs (62) coaxial with the planetary axles (16).

33. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein the adjusting mechanism operatively situated between the planetary gear carrier (18) and the planetary gears (8) are continuously feedable set screw (64) inserted in the planetary gear carrier (18).

34. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein at least one of the planetary gears (8) and the ring gear (14) have an incision (46, 48).

35. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 34, wherein the incision (46) in the planetary gears (8) is designed revolving with rotational symmetry.

36. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 34, wherein several peripherally spaced incisions (46) are located in the planetary gears (8).

37. (PREVIOUSLY PRESENTED) The planetary gear train according to claim 20, wherein a reduction ratio of the gear train has a value which is less than or equal to twelve.

PAGE 5/34 * RCVD AT 4/24/2009 4:36:45 PM [Eastern Daylight Time] * SVR:USPTO-EFXRF-5/33 * DNIS:2738300 * CSID:603 226 7499 * DURATION (mm-ss):07-14